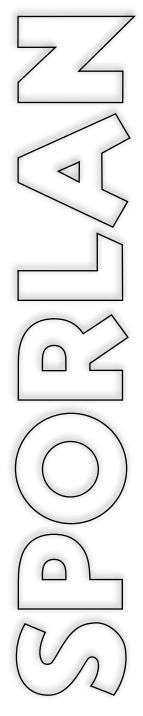


# 3-Way Valves

Installation and Servicing Instructions







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# 3-Way Heat Reclaim Valves

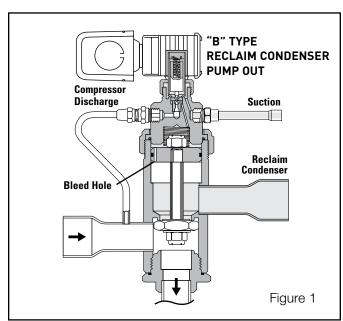
B5D, 8D, 12D, 16D, 8D-HP, 12D-HP, SB5D, S8D, S12D, S16D, S8D-HP, S12D-HP

### **OPERATION**

### "B" Type

### Normal (Outdoor) Condenser - De-energized

See Figure 1. With the pilot valve de-energized, high side pressure is prevented from entering the cavity above the piston assembly. At the same time, the upper pilot port is open to suction pressure. The resulting pressure differential across the piston moves the piston-seat assembly to close the reclaim condenser port (upper main port). In this mode, the refrigerant flows to the normal condenser. The pilot valve opens the cavity above the piston to suction. This allows the reclaim condenser to be pumped out through a small bleed hole in the piston. The pump out process reduces the reclaim condenser to suction pressure. Once suction pressure is reached, the flow through the bleed hole in the piston stops. There is no high to low side bleed with continued operation in the normal condenser mode. For a more efficient pump out of the reclaim condenser, a normally open solenoid valve can be added to the lowest physical location of the reclaim coil to remove liquid.



### "C" Type Normal (Outdoor) Condenser - De-energized

With the pilot valve de-energized, high side pressure is prevented from entering the cavity above the piston-seat assembly. At the same time the upper pilot port is opened to suction pressure. The resulting pressure differential across the piston moves the piston-seat assembly to close the reclaim (upper) main port, thereby eliminating high to low side bleed and the resulting capacity loss with the system in the normal condenser mode.

### "B" and "C" Type Reclaim (Reheat) Condenser - Energized

When the pilot valve is energized, high side pressure is permitted to flow through the lower pilot port at the same time the

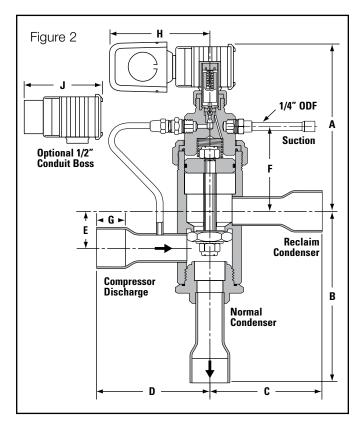
upper pilot port is closed to suction. High side pressure build up on top of the piston moves the piston-seat assembly to close the normal condenser port and open the reclaim (upper) main port. With the upper pilot port closed, there is no high to low side bleed and resulting capacity loss with the system in the reclaim mode.

Valves using the MKC-1 coil may be used on fluids or gases where the temperature does not exceed  $240^{\circ}F$  ( $116^{\circ}C$ ), while the valve ambient is  $120^{\circ}F$  ( $49^{\circ}C$ ). The coil is Underwriters Laboratory Class F rated.

### INSTALLATION

### **Piping Suggestions**

Valves must be installed in a horizontal or vertical position with the coil level with or above the valve body. Install Heat Reclaim Valves so the connections are in the proper flow direction as shown in Figure 2.



## **DIMENSIONS - Inches (mm)**

Туре	Port Size	Connection ODF Solder Inches	A	В	С	D	E	F	G	Н	J
(S)B5D5B (S)B5D5C	5/8	5/8	5.00 (127)	4.36 (111)	1 -	3.22 (82)	0.79 (20)	3.84 (98)	0.50 (13)	2.94 (75)	
(S)8D7B(-HP) (S)8D7C(-HP)	3/4	7/8	5.18	5.06 (129)	3.44	3.44	1.13	2.63	0.75 (19)		
(S)8D9B(-HP) (S)8D9C(-HP)	3/4	1-1/8	(132)	5.13 (130)	(87)	(87)	(29)	(67)	0.91 (23)		
(S)12D11B(-HP) (S)12D11C(-HP)		1-3/8							0.97 (25)	2.89 (73)	1.64 (42)
(S)12D13B(-HP) (S)12D13C(-HP)	1-1/4	1-5/8	6.87 (174)	6.94 (176)		4.19 (106)		4.25 (108)	1.09 (28)		
(S)12D17B(-HP)		2-1/8							1.25 (32)		
(S)16D17B (S)16D17C	2	2-1/8	8.18 (208)	9.53 (242)	1 -	5.47 (139)	3.50 (89)	5.44 (138)	1.25 (32)	3.17 (81)	

Figure 3 shows piping schematics only to illustrate the general installation of the Heat Reclaim Valves. Sporlan recommends that recognized piping references be consulted for assistance in piping procedures. Sporlan is not responsible for system design, any damage resulting from system design, or for misapplication of its products.

Proper support of heat reclaim valves is essential. Concentrated stresses resulting from thermal expansion or compressor vibrations can cause fatigue failure of tubing, elbows and valve fittings. Fatigue failures can also result from vapor propelled liquid slugging, and condensation induced shock. The use of piping brackets close to each of the 3-Way valve fittings is recommended.

# HEAD PRESSURE CONTROL FOR RECLAIM SYSTEMS

When employing heat reclaim on a refrigeration system, the addition of head pressure controls is important not only to maintain liquid pressure at the expansion valve inlet, but also to assure the availability of quality hot gas at the reclaim heat exchanger.

See Sporlan Bulletin 90-30, Head Pressure Control Valves for high and low ambient stability. When low ambient conditions are encountered during fall-winter-spring operation, the Sporlan head pressure controls hold back liquid refrigerant so a portion of the condenser surface is inactive. This results in a rise in condensing pressure.

NOTE: To calculate additional charge required for head pressure control, see Bulletin 30-20, or Bulletin 90-30-1.

#### TYPICAL PIPING SCHEMATICS Figure 3 Check **PARALLEL SERIES** Normal Normal Valve Condenser Condenser **CONDENSERS CONDENSERS** 3-Way Heat **Normally Closed** Reclaim Solenoid Valve **Reclaim Valve** 3-Way Heat Condenser Reclaim Valve Check 5 ② (▼) Check Valve Valve 3>< Check Reclaim Condenser to suction Valve (optional) Normally Open Solenoid Valve (XPM) (optional) **<**(4) Normally Open Solenoid Valve (XPM) A8 Pressure A9 Hot Gas **A9 Hot Gas A8 Pressure Regulating Valve Bypass Regulator Regulating Valve Bypass Regulator** Receive Receive (3) > 0 Compressor Compressor

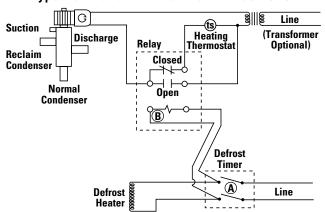
- ① Use normally open solenoid valve and piping if pump out is required and "C" model Heat Reclaim Valve is used, see note 4. Or, if a normally closed valve can be used then use XSP-10. It has a 0.031" orifice, this may eliminate need of cap tube or Sporlan Restrictor 2449-004.
- ② This check valve required if lowest operating ambient temperature is lower than evaporator temperature.
- ③ Restrictor 2449-004 may be required to control pump out rate on inactive condenser.
- Pilot suction line must be open to common suction whether or not Heat Reclaim Coil is installed at time of installation and regardless of Heat Reclaim Valve model/type.
- § Proper support of heat reclaim valves is essential. Concentrated stresses resulting from thermal expansion or compressor vibrations can cause fatigue failure of tubing, elbows and valve fittings. Fatigue failures can also result from vapor propelled liquid slugging, and condensation induced shock. The use of piping brackets close to each of the three way valve fittings is recommended.

### **TYPICAL WIRING SCHEMATICS & ELECTRICAL CONSIDERATIONS**

### Single Compressor Systems

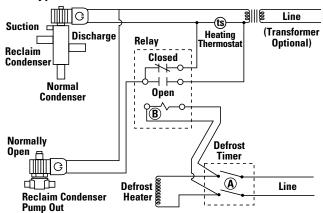
Wiring of single compressor units may require the use of relays to prevent high to low side bleed during periods when the compressor is not circulating refrigerant. This is necessary to prevent refrigerant migration to the low side of the system where it may condense into liquid and damage system components when the compressor restarts.

### "B" Type - OFF CYCLE ELECTRIC DEFROST SYSTEM



Defrost timer (A) energizes relay coil (B) to close normally open contact. This bypasses the (ts) when defrosting to energize the heat reclaim coil and shut off the valve's bleed to suction.

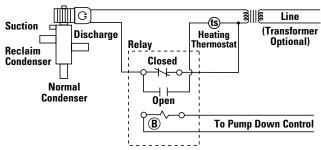
### "C" Type - OFF CYCLE ELECTRIC DEFROST SYSTEM



Defrost timer (A) energizes relay coil (B) to close the normally open contacts. This bypasses the (ts) when defrosting to close the pump out solenoid and shut off the valve's pump out to suction.

It is always a good idea to check with the equipment manufacturer for recommendations on the application of heat reclaim on the specific type of equipment. It is necessary to adhere to local piping and electrical codes that apply to each application. For complete product information, refer to Bulletin 30-20.

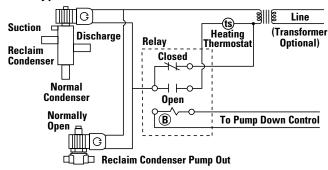
# "B" Type - PUMP DOWN or ELECTRIC DEFROST with PUMP DOWN (unit off)



When the pump down control is energized, relay coil (B) is energized. This allows the heating (ts) to control the 3-Way Heat Reclaim Valve operation.

When the pump down control is de-energized, the normally closed contact is closed. This bypasses the heating (s) to energize the coil and shut off the valve's bleed to suction.

### "C" Type - PUMP DOWN SYSTEM (unit off)



When the pump down control is energized, relay coil  $\textcircled{\textbf{B}}$  is energized. This allows the heating  $\textcircled{\textbf{s}}$  to control the 3-Way Heat Reclaim Valve operation.

When the pump down control is de-energized, the normally closed contact is closed. This bypasses the heating (ts) to energize the reclaim condenser pump out solenoid and shut off the valve's pump out to suction.

### **SERVICE**

### **Typical Malfunctions**

There are only three possible malfunctions: 1. Coil burnout. 2. Failure to shift to Reclaim Mode. 3. Failure to shift to Normal Mode. Each is discussed below:

### **COIL BURNOUT**

Coil burnouts are extremely rare unless caused by one of the following:

- 1. Improper electrical characteristics.
- 2. Continuous over-voltage, more than 10%.
- 3. Under-voltage of more than 15%. This applies only if the operating conditions are such that the reduced MOPD causes stalling of the plunger, which results in excessive current draw.
- 4. Incomplete magnetic circuit due to the omission of the

plunger assembly during reassembly.

- 5. Mechanical interference with movement of plunger which may be caused by a deformed enclosing tube.
- 6. Coil energized while not installed on a valve.

#### **FAILURE TO SHIFT TO RECLAIM MODE**

- 1. Coil burnout (see coil burnout at left).
- 2. MOPD greater than specifications.
- 3. Restricted high pressure pilot connection.
- 4. May not have allowed sufficient time to pump out the reclaim coil while in the normal mode.

#### **FAILURE TO SHIFT TO NORMAL MODE**

- 1. Stray voltage holding plunger up.
- 2. Restricted, closed service valve, or capped suction connection on pilot.

For a list of replacement part kits, see table on page 13.

# 3-Way Split Condenser Valves

8D-SC, 12D-SC, 16D-SC, S8D-SC, S12D-SC, S16D-SC

The Split Condenser Valves are a relatively simple modification of the standard Heat Reclaim Valves. Figure 7 shows that the Split Condenser Valves only use the upper port; the lower port is always open. The location of the seat disc has been optimized such that when the valve is in the two condenser mode, the flow is split evenly between the two condensers.

The pilot valve is different from the standard heat reclaim valve. It is normally open to high pressure.

### **OPERATION**

### "B" and "C" Type

### Two Condenser Mode - De-energized

See Figure 4. With the pilot valve de-energized, high side pressure ① is permitted to flow through the lower pilot port at the same time the upper pilot port is closed to suction ②. High side pressure ① built on top of the piston moves the piston-seat assembly down to evenly split the flow between the two condensers. The piston-seat assembly is held in place by a plate located in the Condenser A connection. This plate is designed to limit restriction through that port. With the upper pilot port closed, there is no high to low side bleed and no resulting capacity loss with the system in the two condenser mode.

Valves using the MKC-1 coil may be used on fluids or gases where the temperature does not exceed 240°F (116°C), while the valve ambient is 120°F (49°C). The coil is Underwriters Laboratory Class F rated.

**NOTE**: If the pressure in your idle Condenser B is less than the suction pressure that the suction connection of the valve is connected to, you may have charge migration back into your idle Condenser B. A small check valve in the suction pilot line can be used to prevent this.

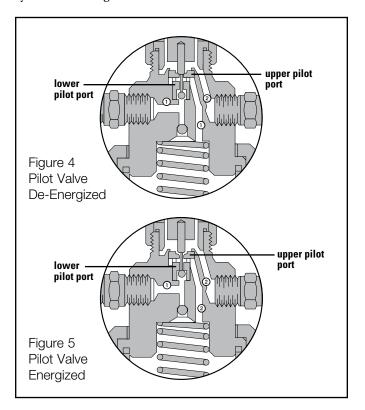
### "B" (Bleed) Type Single Condenser Mode - Energized

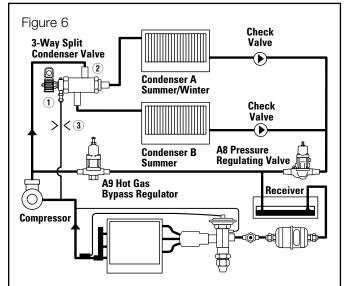
See Figure 5. With the pilot valve energized, high side pressure ① is prevented from entering the cavity above the piston-seat assembly. At the same time the upper pilot port is opened to suction pressure ②. This opens the cavity above the piston-seat assembly to suction pressure ②. The cavity below the piston is exposed to high pressure. The resulting pressure differential across the piston moves the piston-seat assembly up to close the upper (Condenser B) port. We use a solid piston ring on the piston, thereby eliminating high to low side bleed around the piston. When the upper pilot port opens, Condenser B is pumped out through a small bleed hole in the piston. When Condenser B has been pumped out and reduced to suction pressure, all flow ceases, thus eliminating high to low side bleed and the resulting capacity loss that may occur with the system in the single condenser mode.

### "C" (No Bleed) Type Single Condenser Mode - Energized

See Figure 5. With the pilot valve energized, high side pressure ① is prevented from entering the cavity above the piston-seat assembly. At the same time the upper pilot port is opened to suction pressure ②. This opens the cavity above the piston-seat

assembly to suction pressure ②. The cavity below the piston is exposed to high pressure. The resulting pressure differential across the piston moves the piston-seat assembly up to close the upper (Condenser B) port. A solid piston ring is used on the piston thereby eliminating high to low side bleed around the piston and the resulting capacity loss that may occur with the system in the single condenser mode.





- Pilot suction line must be open to common suction regardless of Split Condenser Valve model/type.
- ② Proper support of the Split Condenser Valves, especially the larger models, is essential because of possible vibrations of the piping and/or their location on the compressor racks. (For example, a bracket located 2-3" from the 16D Discharge and Condenser B socket solder joints is recommended.)
- ③ Restrictor, part no. 2449-004, may be used to control pump out rate on inactive condenser.

### INSTALLATION

### **Piping Suggestions**

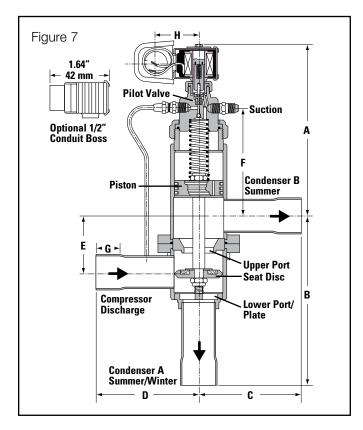
Valves must be installed in a horizontal or vertical position with the coil level with, or above the valve body. Install Split Condenser Valves so the connections are in the proper flow direction as shown in Figure 7.

Figure 6 shows a piping schematic to illustrate the general installation of the Split Condenser Valves. Sporlan recommends that recognized piping references be consulted for assistance in piping procedures. Sporlan is not responsible for system design, any damage resulting from system design, or for misapplication of its products.

Proper support of Split Condenser Valves, especially the larger models, is essential because of possible vibrations of the piping and/or their location on the compressor racks. (For example, a bracket located 2 to 3 inches from the 16D Discharge and Condenser B socket solder joints is recommended.)

### DIMENSIONS - Inches (mm)

Туре	Port Size	Connection ODF Solder Inches	A	В	С	D	E	F	G	Н
(S)8D9B-SC	3/4	1-1/8	5.18 (132)	5.13 (130)	3.44 (87)	3.44 (87)	1.13 (29)	2.63 (67)	0.91 (23)	
(S)12D11B-SC	1 1/4	1-3/8	6.87	6.94	4.19	4.19	2.38	4.25	0.97 (25)	2.94
(S)12D13B-SC	1-1/4	1-5/8	(174)	(176)	(106)	(106)	(60)	(108)	1.09 (28)	(75)
(S)16D17B-SC (S)16D17C-SC	2	2-1/8	8.18 (208)	9.53 (242)	5.47 (139)	5.47 (139)	3.50 (89)	5.44 (138)	1.25 (32)	



# 3-Way Hot Gas Defrost Valves

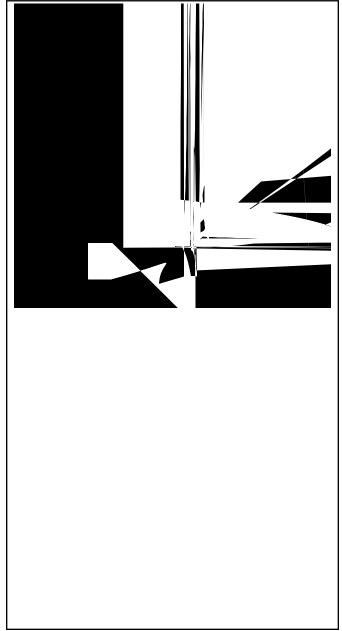
10G79B, 10G711B, 10G711C

### **INSTALLATION**

**CAUTION:** The valve body must be wrapped with a wet cloth while soldering to avoid damage to synthetic internal parts. While the valve's steel body construction will help avoid overheating of internal parts due to conduction, convection of heat up through the tubing and valve body while making bottom connections can damage the internal parts. If possible bench assemble the valve utilizing stub tubes in a horizontal plane. Cool the valve body thoroughly after making each connection.

### **DIMENSIONS - Inches (mm)**

Туре	A	В	C	D	E	F
10G79B	1-1/8 (29)	.91 (23)	6.00 (152)	1.81 (46)	9.79 (249)	
10G711B	1-3/8 (35)	.97 (25)	5.87 (149)	1.74 (44)	9.79 (249)	1.56 (40)
10G711C	1-3/8 (35)	.97 (25)	5.16 (131)	1.72 (44)	10.62 (270)	



### SERVICE

Service parts for these valves may be ordered through your local Sporlan wholesaler.

If a pilot assembly kit is required, note that they – unlike the internal parts kits – are not interchangeable. For a list of replacement part kits, see table on page 13.

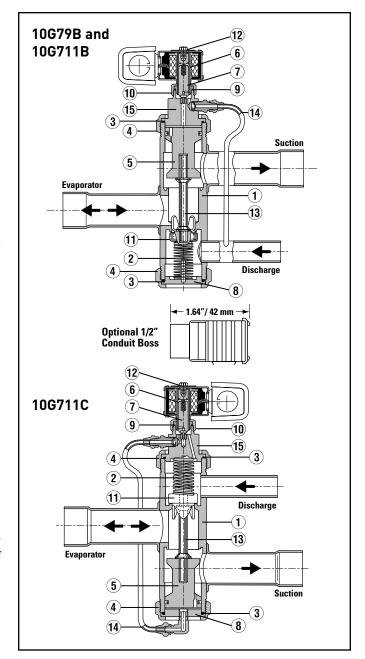
### DISASSEMBLY

- 1. Disconnect the coil lead wires and remove the coil assembly ①.
- 2. Remove the enclosing tube locknut ① to inspect the plunger assembly ⑦.
- 3. Loosen and disconnect the pilot tube (4) at the flare nut.
- 4. Loosen the suction end locknut ④ using a pipe wrench or other suitable wrench. Remove the pilot assembly ⑤ from the body.
- 5. Loosen the discharge end lock nut ④, but use care to prevent damaging the parts. Remove the locknut ④, bottom cap assembly ⑧, valve spring ② and seat assembly ①.
- 6. Slide the piston (5) up and out of the valve body bore.
- 7. Inspect the piston 5 and seat assembly 11 for damage. Replace all tetraseals or gaskets 3 9 if valve has been in service.

# NOTE: OIL TETRASEALS and GASKETS BEFORE INSTALLATION.

### REASSEMBLY

- As a unit, install from the bottom of the body assembly ①:
   10G79B, 10G711B pushrod ③, seat assembly ①, valve spring ②, tetraseal ③, bottom cap assembly ⑧.
   10G711C pushrod ③, piston ⑤, bottom cap assembly ⑧.
   Check center shaft adjustment by looking into the body from the top. Push down on the pushrod ③ to realign if necessary.
- 2. Install the bottom locknut ④ and tighten with a pipe wrench or torque wrench to 50-60 foot-pounds torque. This must be leak tight. Do not overtighten.
- 3. From the top of the body assembly ① install: 10G79B, 10G711B piston ⑤ in body bore, compress the piston ring and slide the piston ⑤ all the way into the bore. 10G711C seat assembly ①, valve spring ②.
- 4. Install the pilot assembly (5) on the body assembly (1) and tighten the top locknut (4) with a pipe wrench or torque wrench to 50-60 foot-pounds torque. This must be leak tight. Do not overtighten.
- 5. Connect the pilot tube (4) flare nut(s) and tighten to 80 inch-pounds torque. This must be leak tight. Do not overtighten.
- 6. Install the plunger assembly ①, enclosing tube assembly ⑥, and enclosing tube locknut ⑩. Tighten the enclosing tube locknut to 10-15 foot-pounds torque.
- 7. Reinstall the coil assembly ② and connect the lead wires.



Item	Part Name
1	10G79 Body Assembly 10G711 Body Assembly
(2)	Valve Spring
3	Tetraseal (641-6) or Gasket 14000-5*
4	Locknut
(5)	Piston
6	Enclosing Tube Assembly
7	Plunger Assembly
8	Bottom Cap Assembly
9	Tetraseal (641-1) or Gasket 14000-1*
10	Enclosing Tube Locknut
11)	Seat Assembly
(12)	Coil Assembly (MKC-1)
13	Pushrod
14)	Pilot Tube
15)	Pilot Assembly

<sup>\*</sup>Part is available separately

# Type 180 Solenoid Pilot Control Valve

### **FEATURES**

The Type 180 Solenoid Pilot Control offers three outstanding features for refrigeration applications:

- Costs less, more economical to install.
- Only one size.
- Assures positive shut-off of liquid line.

The 180 Solenoid Pilot Control is available as a supplementary device to Sporlan Thermostatic Expansion Valves. It is used in place of large capacity solenoid valves for positive shut-off of liquid lines. Since only one size is necessary it costs less and is more economical to install.

The Solenoid Pilot Control does not directly close the liquid line, but acts on the thermostatic expansion valve causing the expansion valve to close. The 180 is installed in the external equalizer line of the thermostatic expansion valve and has a third 1/4" connection from the liquid line.

### **OPERATION**

To understand the operation of the 180 Solenoid Pilot Control, it is important to review the principle of operation of a thermostatic expansion valve. There are three fundamental pressures involved in the operation of any thermostatic expansion valve. They are: ① bulb pressure; ② evaporator pressure; and, ③ spring pressure, as shown in Figure 8.

The bulb pressure acts on one side of the diaphragm tending to open the valve. The evaporator pressure acts on the opposite side of the diaphragm tending to close the valve, and the spring pressure assists in closing the valve. A variation of any one of the three pressures changes the amount of opening of the valve. In normal operation of the valve, these pressures tend to establish a point of equilibrium to maintain a constant

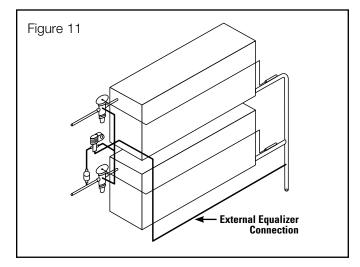
superheat at the outlet of the evaporator. However, when an abnormal pressure is introduced which upsets the equilibrium, another effect may be accomplished. The principle upon which the 180 Solenoid Pilot Control influences the expansion valve action is by the creation of a pressure under the valve diaphragm, which is higher than the bulb pressure. This is how it is accomplished.

The Type 180 has two ports in the valve body — one high pressure and one low pressure. When the solenoid coil is energized, the plunger moves upward sealing off the high-pressure port. With the high pressure excluded from the pilot control, true suction pressure acts on the underside of the expansion valve diaphragm through the equalizer line and the low pressure port. This is illustrated in Figure 9. When the solenoid coil is de-energized, the low pressure port is closed, thereby closing the equalizer line from the valve to the suction line. The high pressure port is open and liquid line pressure is applied to the underside of the thermostatic expansion valve diaphragm. This high-side pressure instantly overom tpressure porressure acts onuideaton.

Upon re-energizing the solenoid coil, the low pressure port of the 180 opens and allows instantaneous relief of pressure under the thermostatic expansion valve diaphragm and normal operation of the thermostatic expansion valve.

### **APPLICATION**

The Type 180 Solenoid Pilot Control may be connected to any number of thermostatic expansion valves as large as those nominally rated at 132 tons on Refrigerant 134a and 180 tons on Refrigerant 22. Thus one Solenoid Pilot Control simultaneously controls the action of all expansion valves on one evaporator or system of evaporators. See Figure 11.



NOTE: Since the thermostatic expansion valve Types V and W have an extended neck between the thermostatic element and the valve body, a relatively large quantity of liquid refrigerant resides at this point during the off-cycle (Type 180 de-energized). Upon re-energizing the Type 180 control this liquid must boil off if the TEV is installed so that the "neck" volume is NOT free draining. The consequence of such an arrangement is chilling of the thermostatic element and possible gas charge condensation or migration.

Therefore, we recommend that Types V or W thermostatic expansion valves (with gas or Type "P" thermostatic charges), when installed with the Type 180 solenoid pilot control, be placed in an upright position and at a point above the suction line so that the volume under the diaphragm is free draining.

The Solenoid Pilot Control is actuated in the same manner as a liquid line solenoid valve, e.g. with a thermostat, a pressure switch, or by manual control. Since the Type 180 design requires that the liquid line pressure during the off-cycle be slightly higher than the expansion valve bulb pressure, certain types of applications require special precautions.

### **Comfort Cooling Applications**

Sporlan thermostatic expansion valves for comfort cooling applications are generally supplied with Type P air conditioning charged thermostatic elements. These charges limit the maximum operating or opening pressure during the off-cycle, as well as during the running cycle. The table lists these standard maximum operating pressures at an approximate 60°F (16°C) saturated evaporator temperature.

Refrigerant	Standard Maximum Operating Pressure
12, 134a	50 psig (3.5 bar)
22	90 psig (6.2 bar)

Therefore, during the off-cycle the receiver ambient temperature needs to be only slightly higher, 65°F (18°C) or more, to maintain a higher liquid line pressure and to ensure satisfactory Solenoid Pilot Control operation.

# Commercial or Low Temperature Refrigeration Applications

This type of application requires that the expansion valve and sensor bulb be installed within the refrigerated space, so the bulb temperature will be the same as the evaporator during the off-cycle. Not only will this prevent compressor flood-back on start-up, but it will help ensure a liquid line pressure higher than the bulb pressure to keep the expansion valve closed tight during the off-cycle.

### **Comfort Cooling Application (Intermittent Operation)**

In certain locales during mild weather, wide variations between night and day temperatures can create unusual system pressure conditions. This occurs when the receiver, hence the liquid line pressure, falls below the maximum thermostatic expansion valve operating pressures listed above. This is liable to occur when the receiver is located on the outside of the building or is subjected to unusually low ambient temperatures. Proper attention to receiver location can minimize the possibility.

### Other Applications

On other applications where the Solenoid Pilot control may be applied, care must be exercised to maintain the correct liquid line – expansion bulb pressure relationship. If the expansion valve bulb is located near heating coils or any other location warmer than the receiver location, trouble may occur. During the off-cycle, the bulb pressure for expansion valves with thermostatic charges other than the Type P air conditioning charge will rise and open the valve.

This will tend to flood the evaporator; and, if the compressor is on pressure control, cause it to run for a short period of time when cooling is not required. Therefore, proper attention to component location is important to ensure correct operation of the Type 180 Solenoid Pilot Control.

### Recommendations

Where the unusual conditions exist as previously described, and where normal location precautions are not feasible, Sporlan recommends that a conventional liquid line solenoid valve be installed instead of the Type 180 Solenoid Pilot Control.

All refrigeration and air conditioning systems should be protected from moisture and other system contaminants by the Sporlan Catch-All® Filter Drier. When using the Type 180 Solenoid Pilot Control, the expansion valve is used to shut off the liquid line in place of a standard liquid line solenoid valve. Therefore, it is essential that the system be free of these contaminants which might prevent the thermostatic expansion valve from seating tight.

On double-ported thermostatic expansion valves, such as Sporlan's Types V and W, the chance of dirt causing leakage is increased since either port could be partially blocked open. If this occurs, the leakage would be even greater since both ports would leak because the seating discs operate on the same shaft. Therefore, an adequately sized Catch-All should be installed ahead of the thermostatic expansion valve for complete protection.

The Type 180 Solenoid Pilot Control is not recommended for application with other makes of thermostatic expansion valves. The reason is that Sporlan diaphragm assemblies are specially designed to withstand the admission of high side pressure through the external equalizer connection.

The Type 180 may be installed either upright or on its side. However, it should not be mounted with the coil housing below the valve body.

### INSTALLATION

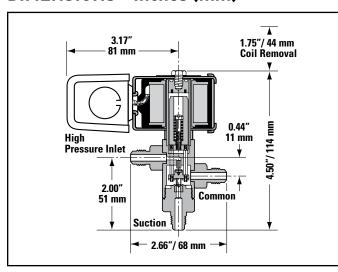
In addition to first cost, flexibility of the Type 180 offers an installation savings. Standard 1/4" soft copper tubing is the only tubing required for the three valve connections as shown in Figure 12. A C-032-F Catch-All Filter-Drier is furnished with each Type 180, since protection of the valve ports from system contaminants is important for proper operation.

#### **Electrical**

For normal summer operation of air conditioning systems, the Type 180 Solenoid Pilot Control may be energized through any actuating device such as a thermostat, a micro-switch, or a manual switch. The compressor can be operated separately through the magnetic starter, wired with a low pressure cutout switch for pump down control.

If it is necessary to shut down a system for a long period, the main liquid line or receiver valve should be closed before de-energizing the entire system at the main electrical service switch. This will protect against compressor floodback if any of the unusual pressure conditions mentioned in the Application section occur during shutdown.

### **DIMENSIONS - Inches (mm)**



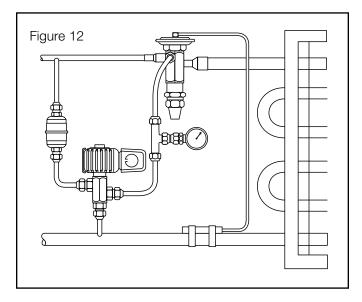
## **SPECIFICATIONS**

MKC-2 Coil

	Туре 180	Standard Coil Ratings		
Connectio	ns - 1/4 SAE Flare	Volts/Cycles	Watts	
MOPD	300 AC (21 bar)	24/50-60		
psi	225 DC (15.5 bar)	120/50-60 208-240/50-60	15	
Maximum Rated	Pressure - psi 500 (35 bar)			

### **SERVICE**

The Type 180 Solenoid Pilot Control can not be taken apart in the field. To properly analyze suspected trouble on an installation, it is advisable to install a pressure gauge by means of a "T" fitting in the external equalizer connection to the thermostatic expansion valve, as shown in Figure 12.



By successively opening and closing the circuit to the Pilot Control only, with the refrigeration system and circulating fans or pumps in operation, variations approximating condensing pressure and suction pressure should be observed on this gauge.

- **A.** If pressure does not build up rapidly to condensing pressure when the Pilot Control is de-energized, proceed as follows:
  - Make certain that the electrical circuit has been broken to the Pilot Control. Since the actuating device is generally at a remote point, we suggest that a means for temporarily interrupting the current be made near the valve installation to facilitate servicing the equipment.
  - **2.** Check to see that no restriction exists and pressure is ahead of both the Solenoid Pilot Control and the C-032-F Catch-All Filter-Drier.
  - 3. Check to ascertain that the low pressure port is closing tightly. A leak in many instances can be detected by a much cooler line from the Pilot Control to the suction line than the liquid inlet line of the control. A further check can be made by removing the suction line connection at the Pilot Control and temporarily plugging it. No leak should be indicated at the suction fitting of the Pilot Control.

- **4.** In the event that full liquid line pressure is available and the bottom port is **NOT** leaking, then it is necessary to replace the Pilot Control valve.
- **B.** If the Pilot Control functions satisfactorily, then the thermostatic expansion valve should be checked. Various causes could be the reason for operation difficulties: pushrod packing leak, seat leak, diaphragm failure, and several others. For a complete service analysis on thermostatic expansion valves, refer to Bulletin 10-11.

### **General Information**

### **INSTALLATION PRECAUTIONS**

- **1.** Do not attempt installation of the valve before pumping the entire system down.
- The solenoid coil must not be energized unless it is installed on the valve. To do so would cause coil to overheat and burn out.
- The solenoid coil should be fused in accordance with local codes.
- 4. If a service valve is installed ahead of a solenoid valve, it should be closed during service procedures. Do not use a solenoid valve as a safety shutoff while making repairs to the system. Be sure there is no liquid in the line between the service and solenoid valves when they are in the closed position that could create dangerous hydraulic pressures.
- **5.** Do not twist the valve assembly by pulling or pushing on the enclosing tube or coil assembly.
- **6.** Do not carry a coil assembly or complete valve by the coil leads. This could damage the coil and cause a coil burnout. Do not carry the valve by the pilot tube. This could cause damage to the flare (SAE) joint creating an external leak.
- **7.** Electrically ground the valve body. Typically this is done through the fluid piping or the electrical conduit.
- **8.** Before energizing the valve, verify that the supply voltage and frequency matches the solenoid coil marking.

### **Soldering Precautions**

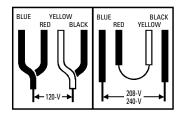
Solder connections on Sporlan Heat Reclaim Valves are copper. Any of the commonly used types of solder are satisfactory with these materials. Regardless of the type of solder used, it is important to avoid over-heating the valve. **Extreme care should be taken when brazing connections to avoid damage to internal synthetic parts.** The torch tip should be large enough to avoid prolonged heating of the connection during the brazing operation. Overheating can also be minimized by directing the flame away from the valve body. **Cool body thoroughly after making each connection.** As an extra precaution, a wet cloth may be wrapped around the valve body during the soldering operation.

**CAUTION:** When installing 1/4" flare pilot line connections, flare body **MUST** be supported with another wrench. On the

5D valve, the pilot valve body **MUST** be supported when tightening the 1/4" flare connection. **Failure to do either of the above may cause leaks or valve damage**.

### Wiring

Check the electrical specifications of the coil to be sure that they correspond to the available electrical service. See page 5 for typical wiring schematics for single compressor systems.



The 1/2" BX conduit connection of junction box on the coil of Heat Reclaim Valves may be rotated to any position by loosening the coil screw. Valves with four-wire dual voltage coil have a wiring diagram decal (see diagram above) on the coil housing or bracket. This illustrates which wires to connect for either 120, 208, or 240 volt operation. Wiring and fusing (when used) must comply with prevailing local and national wiring codes and ordinances.

### TRANSFORMER SELECTION

Coil	24v/5	24v/50-60c   120v/50-60c   3		240v/	50-60c	Transformer Rating Volt-Amperes	
Kit		(	Current	Ampere	For 100% of Rated MOPD		
	In-rush	Holding	In-rush	Holding	In-rush	Holding	of Valve
MKC-1	1.90	0.63	0.39	0.14	0.19	0.09	60
MKC-2	3.10	1.40	0.60	0.26	0.31	0.13	100

### **TEMPERATURE RATINGS**

Basic Valve Type	Coil Type	Ambient Temp. Ratings	Max. Fluid Temp. Rating	Min. Fluid Temp. Rating
8D, 12D, 16D (12-03 & after), 8D-HP, 12D-HP, S8D, S12D, S8D-HP, S12D-HP, 10G	MKC-1	120°F 49°C	240°F 116°C	-40°F -40°C
16D (12-03 & before), 180	MKC-2	120°F 49°C	240°F 116°C	-40°F -40°C

Suitable for use with all Halogenated refrigerants.

### Replacement Part Kits for 8D, 12D, 16D, 8D-SC, 12D-SC, 16D-SC, 8D-HP, 12D-HP, S8D, S12D, S16D, S8D-HP, S12D-HP, 10G

The parts kit for the 3-Way Heat Reclaim Valves include the piston assembly. This requires separate kits for Bleed "B" and Non-Bleed "C" versions of the 3-Way Valves.

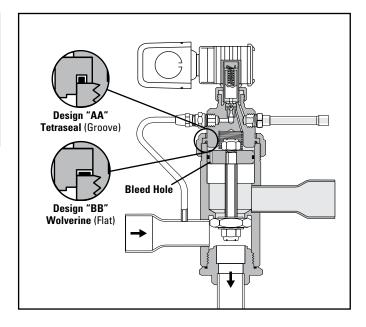
**CAUTION:** If you are replacing the internal parts on an older "B" style valve with split piston rings, you must use the existing piston and piston rings. **Do not use the new piston assembly supplied with your parts kit.** (See page 13.)

3-Way Valve	Manufactured with Split Cast Iron Piston Rings and Honed Body Bore
8D7B	7-96 & Earlier
8D9B	7-96 & Earlier
12D11B	1-94 & Earlier
12D13B	1-94 & Earlier
16D17B	4-91 & Earlier

The parts kit contains two different gaskets for the pilot valve. Which ones are used depends on the pilot valve design.

The Tetraseal, part #641-6 is a flexible gasket and is used on design "AA" valves. These valves can be identified by the square groove on the bottom of the pilot valve.

The bottom of the pilot valve on the design "BB" valve is machined flat to accommodate the metal "Wolverine" gasket part #14000-5.



# **Replacement Parts Kits**

(S)8D7B / 8D9B(-HP) (S)8D7C / 8D9C(-HP) (S)8D9B-SC (S)8D9C-SC (S)12D11B / 12D13B (-HP)	KS-8DB KS-8DC KS-8DB-SC KS-8DC-SC	s / Split Condenser Valves
(S)8D7C / 8D9C(-HP) (S)8D9B-SC (S)8D9C-SC	KS-8DC KS-8DB-SC	1 Pilas Parks Cardons
(S)8D9B-SC (S)8D9C-SC	KS-8DB-SC	1 Dilat Dady Carlot
(S)8D9C-SC		
	K6 0UL 6L	1 Pilot Body Gasket 1 Stem and Seat Assembly
(S)12D11B / 12D13B (-HP)	K3-0DC-3C	1 Piston Assembly*
	KS-12DB	1 Lower Body O-Ring
(S)12D11C / 12D13C(-HP)	KS-12DC	1 Lower Body Gasket (8D ONLY) 1 Upper Valve Seat (12D & 16D ONLY)
(S)12D11B-SC / 12D13B-SC	KS-12DB-SC	T Opper valve Seat (12D & 10D ONLT)
(S)12D11C-SC / 12D13C-SC	KS-12DC-SC	
16D17B	KS-16DB	
(S)16D17B (12-03 and after)	KS-16DB-A	1 Stom and Scot Accombly
16D17C	KS-16DC	1 Stem and Seat Assembly 1 Upper Valve Seat
(S)16D17C (12-03 and after)	KS-16DC-A	1 Piston Assembly*
16D17B-SC	KS-16DB-SC	1 or 2 Lower Body O-Ring
(S)16D17B-SC (12-03 and after)	KS-16DB-SC-A	(16D Valves dated 6-06 and after are welded construction. Stem and Seat Assembly Kits are not available.)
16D17C-SC	KS-16DC-SC	Seat Assembly Kits are not available.
(S)16D17C-SC (12-03 and after)	KS-16DC-SC-A	
Pilot Assembly		1 Enclosing Tube Tetraseal
(S)8D & 12D (-HP)	KS-8D/12DP	1 Coil Housing Screw
(S)8D-SC & 12D-SC	KS-8D/12DP-SC	1 Pilot Valve Assembly
16D	KS-16DP	
(S)16D (12-03 and after)	KS-16DP-A/KS-S16DP-A	1 Coil Housing Screw
16D-SC	KS-16DP-SC	1 Pilot Valve Assembly
(S)16D-SC (12-03 and after)	KS-16DP-SC-A/KS-S16DP-SC	
	Hot Gas	Defrost Valves
10G79B		1 Enclosing Tube Gasket, 2 Pilot Assembly Gaskets
10G711A, B, C	KS-10G	1 Plunger Assembly, 1 Pushrod
10G713		1 Seat Assembly, 1 Piston Assembly, 1 Valve Spring
Pilot Ass	sembly	2 Pilot Assembly Gaskets
10G-B	KS-10GP-B	1 Pilot Valve Assembly
10G-C	KS-10GP-C	1 Coil Housing Screw, 1 Bottom Plate
	Enclos	ing Tube Kits
10G, (S)16D, (S)16D-SC (12-03 and after)	KE-6	1 Enclosing Tube Assembly
16D/16D-SC (before 12-03)	KE-9	1 Enclosing Tube Gasket 1 Coil Housing Screw
8D/12D, 8D-SC/12D-SC	KE-8D/12D	

<sup>\*</sup> Use existing piston on valves with honed bore. See caution note on page 12.

## **RECOMMENDED TORQUE**

Valve Series	Enclosing Tube Locknut	Pilot Valve Assembly Locknut	Lower Body Locknut	Body Flange Cap Screw	Coil Screw
S8D, S8D-HP, 8D, 8D-SC, 8D-HP, S8D-SC	10-15	60-65	25	_	2.3
S12D, S12D-HP, 12D, 12D-SC, S12D-SC	10-15	60-65	_	15-18	2.3
16D, 16D-SC	30-35	***	_	20-24	2.3
S16D, S16D-SC, 16D, 16D-SC (12-03 and after)	10-15	60-65	_	20-24	2.3
10G	10-15	60-65	60-65	_	2.3
180	_	_	_	_	2.3

<sup>\*\*\*</sup> The 16D pilot assembly is connected to the body with a pipe connection. Apply a light coat of #242 (blue) Loctite to the male pipe threads and torque to 30-60 ft.-lb.

Listed by Underwriters' Laboratories, Inc. Listed by Canadian Standards Association Guide-Y10Z Guide-440-A-0 File No. MH4576 File No. 19953

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